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AN EMPIRICAL APPROACH TO PREDICTING CANNON TUBE EROSION RATE

Rauf Imam

Watervliet Arsenal Watervliet, New York

August 1974

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One difficulty in the wear pred	liction in new car	nnon systems is the uncertain-
ty in its service life criteria. F	or tubes with lo	w erosion rate, the service
I fe is generally fatigue limited, vice life is limited by the accuracy	and in tubes with	n nigh erosion rate the ser-
pends on the extent of erosion at t	he origin of rif	u. IMIS ECCUTECY IN TURN de-
respondence in the bore enlargement	and the accuracy	v of firing is non-existent
It varies from weapon to weapon, an	nd within the wear	pon, it varies between
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Abstract continued

different charges and different projectiles. It is this factor that has rendered/earlier wear prediction equations unworkable. To circumvent this problem, equations have been developed that provide the wear rate of cannon from the internal ballistic data. This rate changes as rounds accumulate on a tube. Two equations have been developed that provide an upper and a lower limit for the wear rate of all cannon. The calculated results agree well with most cannon tubes. It is found that with the help of these equations, wear rate of totally new systems can be predicted to within about 25% of that observed in most cannon with very few exceptions.

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AN EMPIRICAL APPROACH TO PREDICTING CANNON TUBE EROSION RATE

A. R. IMAM



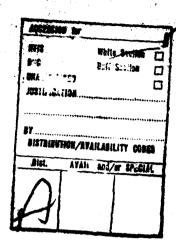
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INTRODUCTION

One of the major problems associated with cannon is erosion of the bore, a problem that has attracted the attention of all classes of workers involved in the development of cannon systems. A somewhat related, but not so intense, problem is that of life expectancy of new cannon. In recent years, due to the high cost of test firing, the need for an educated guess of the wear life of a gun tube has become as urgent as the control of erosion itself. Among the newer cannon systems, such a need was realized when the life of a 155mm M185 tube fell short of the, then desired and hoped for, wear life of the M126 howitzer. This situation repeated itself in the development of the 155mm XM199 and in the 8 inch XM201 cannon tube. The need for a formula to predict wear life of gun tubes has thus become acute. In what follows, a method to alleviate the immediate problem is proposed, with a brief description of the earlier attempts at resolving the problem. It is suggested that the proposed formula for wear rate be considered interim in nature until the development of an analytical expression for such a purpose is realized by understanding the phenomena involved in the erosion of guns.

a. Backgrourd

Unlike the phenomenological and theoretical study of erosion in guns, the prediction of erosion has not been a very popular topic. The earliest attempt seems to be that of Nordheim, et al¹ during World War II, when the erosion rate in vents was calculated with a

Nordheim, L.W., Soodak, H. & Nordheim, G., "Thermal Effects of Propellant Gases in Erosion Vents and Guns" NDRC Report #A-262 OSRD No. 3447.

fair degree of accuracy. This was the most analytical of all attempts that have been made to solve this problem thus far, the present included. Their method consisted of calculating the heat input in conjunction with the bore surface temperature of a vent as well as weight loss and diameter change in the vents, assuming that surface steel melting was the basic mechanism of erosion. These calculations agreed very well with the observed data. However, the method could not be extended to gun tubes primarily because of the lack of understanding of the mechanism of erosion in the gun tubes. In addition, calculations had shown that the bore surface temperatures in most cannon tubes do not reach the melting point of steel, thereby excluding the possibility of postulating general melting in all cannon.

b. Earlier Work

Later, several empirical methods were proposed by the Ballistic Research Labs (BRL) to estimate the wear rate as well as the erosion life of cannon tubes. The two terms may appear similar but are categorically different concepts. The former, that is, the wear or erosion rate, is the only term that can be discussed meaningfully. The latter term involves the criteria of condemnation of a gun tube that differ from weapon to weapon and hence, cannot be considered in any general consideration.

In 1951, Jones and Breitbart² proposed a method to calculate the wear rate in cannon. Although it was claimed to be analytical, in reality it was highly empirical as well as arbitrary. The method

^{2.} Jones, R.N., & Breitbart, S., "A Thermal Theory for Erosion of Guns by Powder Gases", BRL Report #747, 1951.

assumed the existence of steel melting as the predominant factor in the erosion of gun tubes, without calculating the bore surface temperature. It was rather involved and did not produce good results when applied to the present weapons.

Later, Breitbart³ developed a simpler formula from the detailed expression of wear rate, which was highly arbitrary and implied a decrease in wear rate with a rise in pressure above 28,000 psi. In addition, at pressures below 16,000 psi, the equation predicts a decrease in the diameter rather than an increase. Further, the method had the same drawbacks as the earlier detailed one, viz., a lack of agreement between the observed and the calculated data.

Recognizing the above, Frankel and Kruse⁴ from BRL attempted a statistical approach to the problem. Apparently, the method was first applied on United Kingdom cannon. It consists of postulating an empirical expression for bore surface temperature of different cannon using the weight of propellant, the flame temperature and bore diameter, as follows:

$$\theta = \frac{T_0 - 300}{1.7 + 0.38 \, d^{1/2}} \, (\frac{d^2}{c}) \qquad 0.86$$
 (1)

where To is the flame temperature of the propellant

- d the bore diameter
- c the weight of propellant

^{3.} Breitbart, S., "A Simplified Method for Calculating Erosion in Guns" BRL Mem. Report No. 549, 1951.

^{4.} Frankel, J. and Kruse, L., "A Method for Estimating the Service Life of a Gun or Howitzer", BRL Mem. Rpt. 1852 June 1967.

and 0 is the rise in the bore temperature at the origin of rifling.

If w is the wear rate than $\ln \left(\frac{w}{\sqrt{d}}\right)$ is plotted against θ , and

an expression is obtained for w with

$$w = 9.09 \times 10^{-8} \text{ e}^{0.00777}$$
 (2)

A recalculation was proposed for weapons having θ greater than 600°C, resulting in a different value for $\frac{w}{\sqrt{d}}$, namely

The expressions were found by the authors to be unsatisfactory, particularly for howitzers. They considered total service life to be more important than the wear rate, notwithstanding the peril involved in such a venture. Therefore, they suggested a third expression to estimate the service life, namely:

$$L = K e^{a\theta}$$
 (4)

where L is service life and K and a are constants determined statistically by plotting L against θ .

c Discussion of Earlier Work

The last formula cannot be used as a predictive equation since the condensation criteria in a cannon system are generally not determined in advance of substantial test firing. Furthermore, the larger cannon are eften condemned on the basis of fatigue life if

the erosion life is long enough and, hence, under those conditions, the bore surface temperature rise is a meaningless quantity. This aspect of the authors' calculations becomes obvious when the estimated service life of the 105mm M2A1 howitzer of 20,000 rounds and that of the 155mm M1A1 of 15,000 rounds are considered as compared to the actual service life of at 30 rounds for the 105mm M2A1 and 7500 rounds for the 155mm M1A1.

There are three main objections to Frankel and Krause's first two expressions. In the first place, the prediction of a single value of wear rate is almost as bad as the prediction of total service life, because generally, the wear rate changes during the life of a single gun tube and, if the wear rate considered is obtained by dividing the total erosion by total rounds, then the prediction can only be made for the same condemnation limits. The second objection concerns the criteria of condemnation used in arriving at these expressions.

Table I in the BRL Memo Report #1852, provides the wear rate and the service lives of the cannon considered. Below is a sample taken for wear rate and service life from that report with a column of total wear (the product of Wear Rate x Service Life) added for comparison:

Gun	Wear Rate	Service Life	Total Wear
90mm M1	5.32 x 10 ⁻⁵ "/rd.	2,800 rounds	. 149"
90mm M3	7.45 x 10-5	2,060	.149"
90mm M41	28.14 x 10 ⁻⁵	700	. 197"

^{5. &}quot;Hypervelocity Guns and the Control of Gun Erosion" Summary Technical Report of Division I, NDRC Volume 1, OSRD #6, Washington, D.C.1946

Gun	Wear Rate	Service Life	Tota! Wear
105mm M68	75 x 10 ⁻⁵	100	. 075"
175num M113	50 x 10 ⁻⁵	400	. 200''
8" How. Mz	2.5×10^{-5}	6,000	. 150"
8" Gun M1	51.43 x 10-5	700	. 360"

Present weapons do not seem to have similar condemnation limits. Hence, the use of an expression which is developed on these data as a predictive expression for newer weapons, will be incorrect.

The third objection concerns the very nature of the equations. In the first place, the values of θ suggested are grossly misleading. For example, the value for 105mm M2Al How. is 400°C at the origin of rifling. This implies that the tube should not even develop any type of heat affected zone. The fact is that the howitzers develop white layers near the origin of rifling. Further, the maximum temperature indicated in their report for any cannon is 1141°C. It is a fact that melting is observed in some cannon indicating much higher temperatures.

NEW APPROACH

These are the reasons the determination of life estimation equations for gun tubes was left without any concrete starting point. The only course left was to start anew on the problem. In the following sections, a new approach is proposed which, though empirical, still maintains the influences of the ballistic parameters in the same order as is known to exist through experience.

a. Theor Lical Considerations

To develop a precise method for predicting the erosion in an analytical fashion from theoretical calculations requires additional investigative efforts in the fundamentals of gun erosion. Even then

a statistical step will be needed to bridge the final gap of the observed phenomena and the theoretical calculations. The theoretical work would consider calculations of bore surface temperatures, heat input, as well as ballistic parameters such as velocity and projectile engraving forces. A statistical relationship could then be developed for erosion rate in presently existing weapons. This should provide the unknown relating constants between erosion and ballistics that might be applied universally to all weapons. In the absence of such exact data, one has to look for simpler means. In the following sections, such a method is developed.

The ballistic factors that an engineer must consider are the quantity of charge, the pressure, the muzzle velocity and the weight of the projectile. These are not totally independent variables of interior ballistics. They are the apparent influences to which any cannon is subjected. Further, they are functionally related to each other. It would be a mistake to consider these parameters as the complete list of factors giving rise to erosion. In fact, in an exact calculation, these variables will be involved implicitly along with other propellant and cannon-material properties.

Nevertheless, an examination of the erosion rates of different cannon indicates that higher rates of erosion are generally associated with higher muzzle velocity, higher pressure and higher quantitites of charge.

The last factor could be misleading, however, when comparing different caliber cannon. For example, a 105mm M68 heat round has a charge weight of about 11-1/2 lbs. as compared to a weight of 13 lbs. in a 155mm M126 Zone 7 charge. It is well known that the wear rate in the M68 tube is about 400 times as high as in an M126 tube. As a first approximation, therefore, a better choice for incorporation into a wear equation should be the loading density, or perhaps density based on the complete burn out volume of the charge. As the latter involves either some complex calculations or assumptions that may vary from cannon to cannon and charge to charge, for simplicity, therefore loading density is a fair approximation. Pressure, per se, does not seem to have as high an influence on erosion as other factors such as velocity or loading density. The pressure does affect wear indirectly, though, by affecting the velocity and density of combustion gases. One of the most significant factors is the muzzle velocity (MV). In some cases, it has been observed that a difference of 300 fps in MV increases the erosion rate by a factor of 4 to 5. Another important, though not very obvious, factor affecting wear, is the duration of heating during a firing cycle. A rough approximation of this factor is the length of travel divided by the mean velocity (i.e., the muzzle velocity divided by 2).

Once the relative importance of erosion influencing factors is determined, the next step is to correlate them with the erosion rate.

To do so in a direct manner would be time consuming, and would not

add any extra confidence in the end results. If we examine the existing information on the erosion of different weapons, it is observed that cannon can be classified in three categories of severity of erosion. The first is comprised of howitzers with muzzle velocity under 2000 fps and using Ml propellant. These tubes apparently have such low erosion rates that they are condemned on the basis of fatigue. The second contains weapons having M.V. between 2000 fps and 3000 fps and a bore size less than 8 inches. These weapons have moderately high erosion rates and are condemned due to their erosion. The third is comprised of cannon having M.V. greater than 3000 fps or bore size larger than 8 inch. These cannon wear at extremely high rates and the nature of erosion is such that melting of the bore surface may be quite prominent.

b. First Equation

With these observations in mind, it is, therefore, suggested that the wear data over the whole life of one of the cannon in the second classification be used as the starting point, i.e., consider the erosion of one of the cannon as the standard and relate the other tubes through the ratio of their ballistic parameters. The ratios of the factors involved are weighted according to their importance.

As a first approximation, the following is suggested:

$$W_2 = W_1 \cdot \frac{\Delta_2}{\Delta_1} \cdot \frac{p_2^{1/2}}{p_1} \cdot \frac{L_2/v_2}{L_1/v_1} \cdot \frac{v_2^3}{v_1}$$
 (5)

or
$$W_2 = W_1 \cdot \frac{\Delta_2}{\Delta_1} \cdot \frac{P_2^{1/2}}{P_1} \cdot \frac{L_2}{L_1} \cdot \frac{V_2^2}{V_1}$$
 (6)

where, W is the erosion rate,

A is the loading density,

P is the maximum pressure,

V is the muzzle velocity,

and L is the length of travel

The subscript 1 refers to the standard cannon (in this case, the 155mm Howitzer M185) and the subscript 2 refers to the unknown cannon.

To calculate the wear life of a tube, first calculate the average erosion rate of the standard tube at different stages of its erosion life, in terms of discrete percentages of diameter change due to erosion. Ten points (i.e. diameter changes) were selected on the graph of the diameter against the rounds fired on the 155mm M185. For each point, the observed cumulative diameter change was converted to percent of bore diameter. At these points, the cumulative diameter change was divided by the rounds fired to provide W1 (standard).

Using equation (6), W_2 was then calculated for a series of cannon tubes for each percentage diameter change. The percentage diameter changes were then converted into the actual diameter changes for the cannon in question. The diameter changes were divided by the appropriate W_2 , giving the number of rounds against a certain diameter change.

Equation (6) can thus be used to calculate the wear rate at the

origin of rifling, for other artillery and tank cannon if the charges consist of M6 propellant without any coolant additive as 14 the M185.

To carry out a correction for propellant and additive, there are certain difficulties that must be stated first. In the canada tubes of interest, the most common propellants are the M1, M6 and M30 or M30El. M1 and M6 are both single base but M6 is about 200°C hotter whereas M30 or M30El are triple base and are about 350°C hotter than M6. The problem one is faced with is the total lack of any current comparative study of these three propellants from the erosion point of view. Hence, to arrive at any corrective factors, earlier studies had to be used. Two such studies are available; one conducted under the National Defense Research Counities and the other a British Study⁶. Although these studies compared different propellants than those in general use presently, considering the temperature ranges only, the following corrective factors can be obtained:

Propellant factor (PF) for M30 and M30El = $2.5 \times M6$. That is, M30 and M30El propellants are $2.5 \times M6$. Also the PF between M6 and M1 propellants is such that erosion for M6 is 3 times greater than for M1.

PF (M30 and M30E1) = $2.5 \times M6$

PF (M6)

 $= 3 \times M1$

^{5. &}quot;Hypervelocity Guns and The Control of Gun Erosion" Summery Technical Report of Division I, NDRC Volume 1, OSRD #6, Washington, D.C. 1946.

^{6.} Abram, H.H. Williams, T., Allen, K.F., "Examination of Six Q.F. 17 Pr. Gun Barrels Used in Ballistic and Erosion Trials of Propellants of Various Composition and Flame Temperature", Armament Research Establishment Metallurgy Report \$754, AD #31516 Weelwich, U.K. March 1954.

In the suggested equation, the wear rate, W_2 , calculated from W_1 is to be multiplied by 2.5 if the propellant used is M30 or M30E1, and W_2 is to be divided by 3 if the propellant used is M1, since the basic data were developed for M6 propellant.

The Coolant Factor (CF) from the scanty previous data is taken to be 3, i.e., if the charge utilized a titanium dioxide jacket, then W_2 should be divided by 3. If no coolant was used, CF = 1. Therefore, the final form of the erosion equation is

$$W_2 = W_1 \cdot \frac{\Delta_2}{\Delta_1} \cdot \frac{P_2^{1/2}}{P_1} \cdot \frac{L_2}{L_1} \cdot \frac{V_2^2}{V_1} \cdot (PF) \cdot (CF)$$
 (7)

which can be combined to yield

$$W_2 = W_1 . K . \Delta_2 . (P_2)^{1/2} . L_2 . (V_2)^2 . (PF) . (CF)$$
 (8)
where $K = (D_1 . (P_1)^{1/2} . L_1 x V_1^2) -1$

c. Discussion

Considering the data calculated, the very first question to be considered is the wear limits assumed. As far as the calculations are considered, the terminal change of diameter does not necessarily represent the service life of the tube. The terminal values shown in these data represent a fixed percentage of their original nominal bore diameters. Only in the case of the 155mm M185 does it appear to be the limit for total service life. On the question of total service life, as stated earlier, due to varied criteria of condemnation, at this stage, it is impossible to prescribe a wear limit and thus service life of a particular design in terms of rounds fired. In general, the 105mm tubes have a

wear limit of about 0.08 inch, and the 155mm tubes from 0.070 to 0.153 inches. In addition, in the 105mm M137 and the 155mm M126 howitzers, the condemnation limit due to fatigue is reached before the wear limit is reached. For the 175mm tube, the limit has been 0.2 inches and for the 8 inch XM201, it is not yet established, but indications are that it may be between 0.13 and 0.15 inches.

Data were calculated using the ballistic data from Table 1 and Equation 8. They are shown in Table 2 and Figures 1 and 2. The data for the 105mm M137 and for the 105mm M126 with 27 charges are indicative of the general trend for cannon with very low rates of erosion. The wear data for the M126 XM119 charge are not established and this round was abandoned. Nevertheless, the calculated figures agree reasonably well with the scanty firing data reported. For the two inch XM201 rounds, although the wear limits are not yet established, the calculated rate seems to be fairly close to the initial reported firing data.

d. Second Equation

The firing data that indicated a large departure from those calculated were the 105mm XM205, the 155mm XM199 and the 175mm M113. To overcome these differences, equation 3 can be changed to enhance the effect of muzzle velocity. The new equation becomes

$$W_2 = W_1 \quad K^1 \quad (\Delta_2) \quad (P_2^{1/2}) \quad (L_2/V_2) \quad (e^{-2V_2}) \quad (PF) \quad (CF)$$
where $K^1 = \frac{1}{(\Delta_1) \quad (P_1)^{1/2} \quad (L_1/V_1) \quad (e^{-2V_1})}$

TABLE 1. BALLISTIC DATA

		W Wt. of Charge	C Chamber Volume	L Length of travel	p Pressure	V Muzzle Velocity	CF Coolant	PF Propellant	
Weapon/Charge	Bore	(Lbs.)	(Cu. in)	$(In. \times 10^2)$	(KSI)	(1000 ft/sec)	Factor	-	Propellant
M185/M119	155mm	20.4	1150	2.80	30.2	2.245	-	-	Ж
M137/2467 Z7	105144	2.8	153	1.10	36.7	1.621	-	1/3	M
XM205	105mm	4.4	153	1.40	45.	2.170	1-1/3	2.5	M30A1
M68/Heat	105mm	11.5	369	1.86	60.5	3.850		2.5	M30
M126/H4A1 27	155mm	13.2	804	1.16	36.4	1.841	H	1/3	Z
- W126/XW119	155mm	17.5	804	1.16	48.2	2.245	-	2.5	M30A1
XM199/XM123 28	155mm	26.6	1150	2.00	46.0	2.655	1/3	2.5	M30A1
M2 Gun/M119	155	30.9	1596	2.33	45.3	2.800	-	-	*
M113/M86A2 Z3	175mm	57.2	2898	3.52	47.2	3.000	1/3	-	¥
XX201/29	8 Inch	43.6	1950	2.74	39.6	2.530	1/3	2.5	M30A1
92	8 Inch	38.0	1950	2.74	32.5	2.300	1/3	2.5	M30A1

TABLE 2. CALCULATED DATA (EQUATION 8)

155MM M185	Charge M119	105MM M137	Charge M137 Z7
No. of Rds.	Erosion in Inches	No. of Rds.	Erosion in Inches
391	0.045	2416	0.030
812	0.070	5015	0.047
1217	0.088	7521	0.060
1600	0.100	9886	0.068
2000	0.110	12358	0.075
2388	0.117	14754	0.079
2801	0.121	17307	0 .0 82
3198	0.126	19760	0 .08 5
3542	0.130	21887	0.088
4000	0.135	24716	0.091
105MM XM205		105MM M68	Heat-T
No. of Rds.	Erosion in Inches	No. of Rds.	Erosion in Inches
247	0.030	16	0.030
513	0.047	32	0.047
770	0.060	48	0.060
1012	0.068	64	0.068
1265	0.075	79	0.075
1510	0.079	95	0.079
2771	0.082	111	0.082
2022	0.085	127	0.085
2240	0.088	141	0.088
2530	0.091	159	0.091
155MM M126 N	14Al Charge Z7	155MM M126	XM119 Z8
No. of Rds.	Erosion in Inches	No. of Rds.	Erosion in Inches
2960	0.045	174	0.045
6144	0.070	362	0.070
9204	0.088	543	0.088
12112	0.100	714	0.100
15140	0.110	892	0.110
18075	0.117	1065	0.117
21203	0.121	1250	0.121
24208	0.126	1427	0.126
26814	0.130	1580	0.130
30279	0.135	1785	0.135

TABLE 2. CALCULATED DATA (EQUATION 8) (cont)

155MM XM199 Charge XM123 Z8 155MM M2 Gum M19 Normal

No. of Rds.	Erosion in Inches	No. of Rds.	Erosion in Inches
209	0.045	162	0,045
434	0.070	335	0.070
650	0.088	503	0.088
855	0.100	661	0.100
1068	0.110	827	0.110
1275	0.117	987	0.117
1496	0.121	1158	0.121
1708	0.126	1322	0.126
1892	0.130	1464	0.130
2137	0.135	1653	0.135

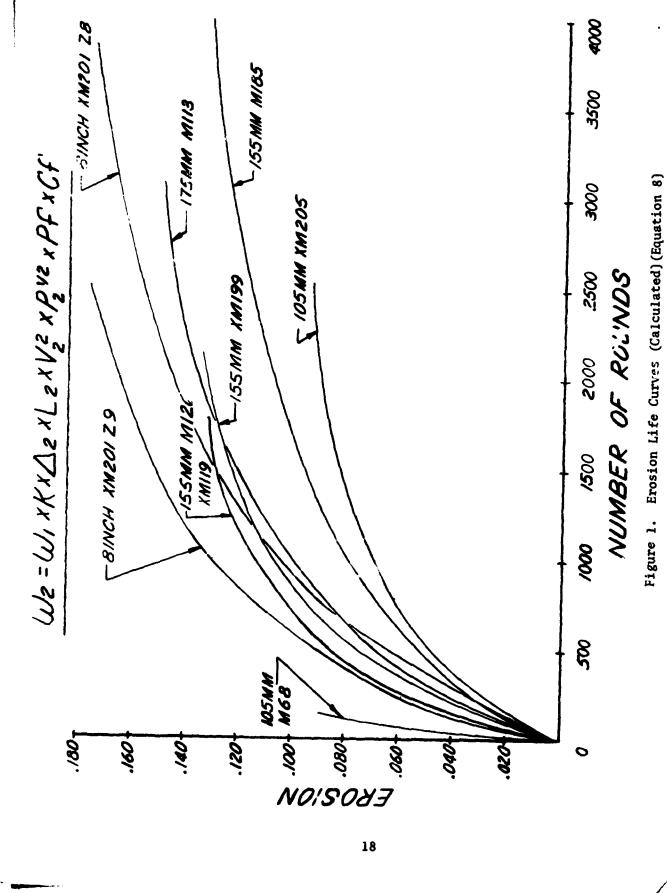
175MM M113 M86A2 Z3

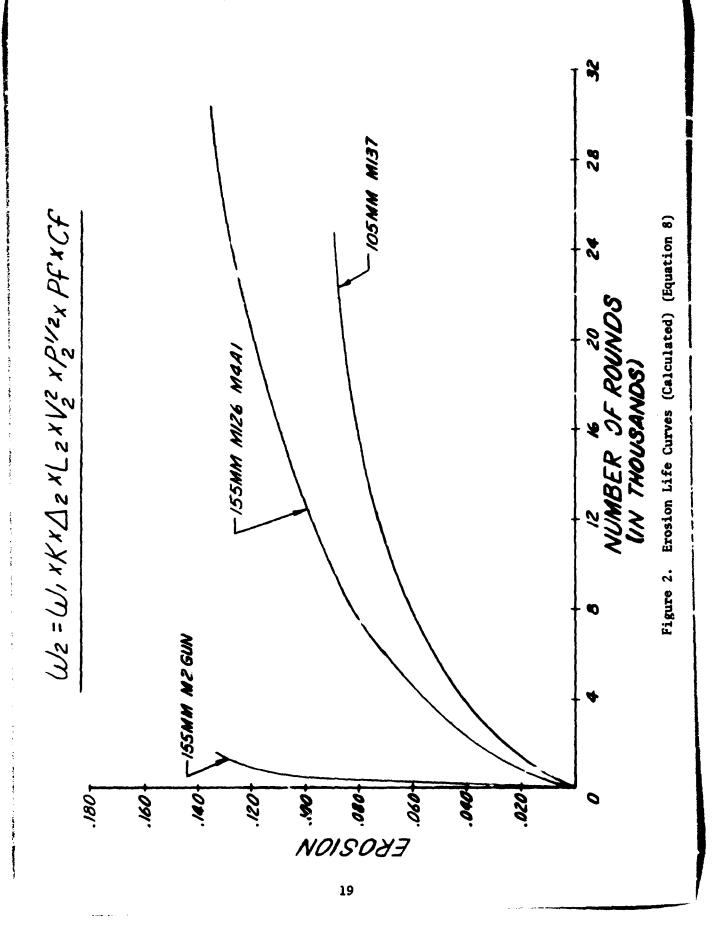
8 Inch XM201 Z9

No. of Rds.	Erosion in Inches	No. of Rds.	Erosion in Inches
303	0.051	245	0.059
628	0.079	508	0.092
942	0.099	762	0.115
1239	0.113	1002	0.131
1548	0.124	1253	0.144
1849	0.132	1496	0.153
2169	0.137	1755	0.159
2476	0.142	2003	0.165
2742	0.147	2219	0.170
3097	0.152	2506	0.177

8 Inch XM201 Z8

No. of Rds.	Erosion in Inches
375	0.059
779	0.092
1168	0.115
1536	0.131
1920	0.144
2292	0.153
2689	0.159
3070	0.165
3401	0.170
3840	0.177





e. Discussion

This equation was used to calculate the data in Table 3 and Figures 3-5. The results obtained are compared to the best available firing data in Table 4. The comparison indicates that the number of rounds for equivalent wear are higher for the M137 and the M126 over that of the first equation. For the XM205, the change is minor, i.e., a slight increase in rounds fo. a given amount of wear. All the others show a decrease in the number of rounds for a given amount of wear. For the XM199, at a diameter increase of 0.150 inches, the corresponding number of rounds are 2000 as against 3200 from the first equation and is much closer to the observed value of 2300 rounds. For the XM201, Zone 9, the value is about 1200 rounds at a diameter change of 0.15 inch whereas for Zone 8, the corresponding number of rounds is about 2200. Although no wear limit has been established as yet, indications are that for Zone 9, the wear limit will be about 0.15 inches. The results of firing of one plated XM201 tube shows the number of Zone 9 rounds against this erosion is about 1500. Similarly, for Zone 8, from firing data of two tubes*, the number of rounds fired for the same wear is 2500. The calculated value with equation (9) is not at all in agreement with the observed value of M68 Heat round where the calculated value is much lower. In fact, the calculated value from equation (8) is closer to the observed value. In the case of the M113 tube, the calculated value using equation (9) is not as high as in equation (8), but is still

^{*}One of the two tubes was chrome plated and the other unplated. At 2500 rounds the two tubes had eroded to about the same extent - 0.15".

TABLE 3. CALCULATED DATA (EQUATION 9)

155mm M185	Charge M119	105mm M137	Charge M67 Z7
No. of Rds.	Erosion in Inches	No. of Rds.	Erosion in Inches
301	0.045	3168	0.030
812	0.070	6576	0.047
1217	0.088	0962	0.060
1600	0.100	12964	0.068
2000	0.110	16205	0.075
2388	0.117	19347	0.079
2801	0.121	22694	0.082
3198	0.126	25911	0.085
3542	0.130	28701	0.088
4000	0.135	32410	0.091
105mm XM205		105MM M68	Charge, Heat T.
No. of Rds.	Erosion in Inches	No. of Rds.	Erosion in Inches
250	0.030	3	0.030
259	0.030	7	0.047
539 808	0.060	10	0.060
1962	0.068	13	0.068
1327	0.075	16	0.075
1584	0.079	19	0.079
1859	0.082	23	0.082
2122	0.085	26	0.085
2350	0.088	29	0.088
2654	0.091	32	0.091
155MM M126	Charge M4A1 Z7	155MM M126	Charge XM119 Z8
No. of Rds.	Erosion in Inches	No. of Rds.	Erosion in Inches
3661	0.045	174	υ.045
7601	0.070	362	0.070
11399	0.088	543	0.088
14984	0.100	714	0.100
18730	0.110	892	0.110
22361	0.117	1065	0.117
26231	0.121	1250	0.121
29949	0.126	1427	0.126
33173	0.130	1580	0.130
37460	0.135	1785	0.135

TABLE 3. CALCULATED DATA (EQUATION 9) (cont)

155MM XM199 Charge XM123 Z8		155mm N2 Gum N19 Normal	
No. of Rds.	Erosion in Inches	No. of Rds.	Erosion in Inches
152	0.045	103	0.045
316	0.070	214	0.070
474	0.088	322	0.088
623	0.100	423	0.100
778	0.110	529	0.110
929	0.117	631	0.117
1090	0.121	740	0.121
1244	0.126	845	0.126
1378	0.130	936	0.130
1557	0.135	1057	0.135

175MM M113 Charge M06A2 Z3 8 Inch 104201 Z9

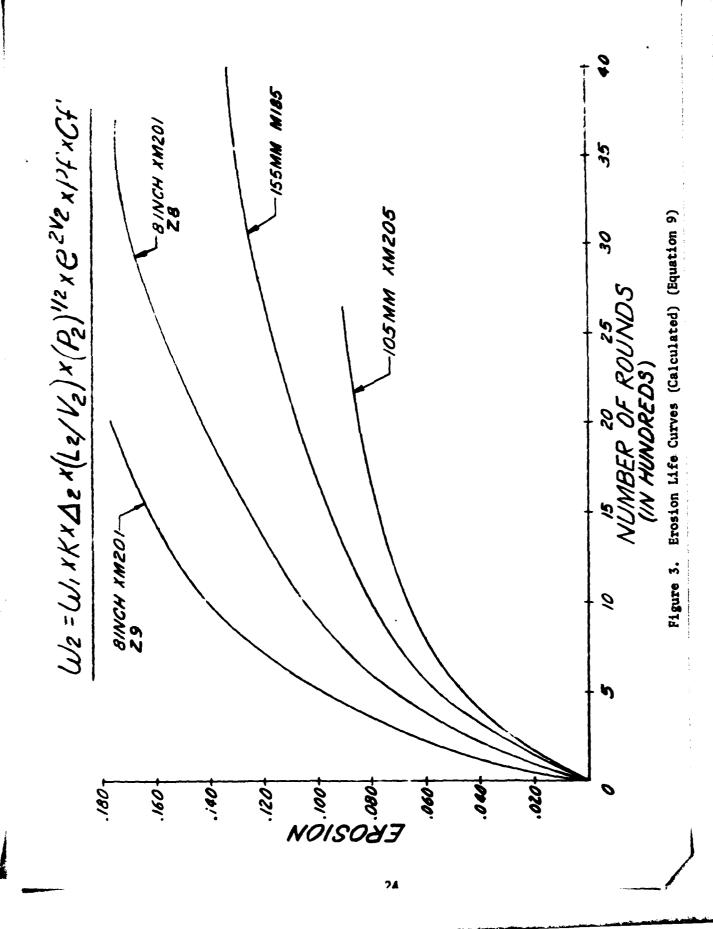
No. of Rds.	Erosion in Inches	No. of Rds.	Erosion in Inches
160	0.051	198	0.059
331	0.079	412	0.092
497	0.099	617	0.115
553	0.113	811	0.131
816	0.124	1014	0.144
975	0.132	1211	0.153
1143	0.137	1420	0.159
1305	0.142	1622	0.165
1446	0.147	1796	0.170
1633	0.152	2028	0.177

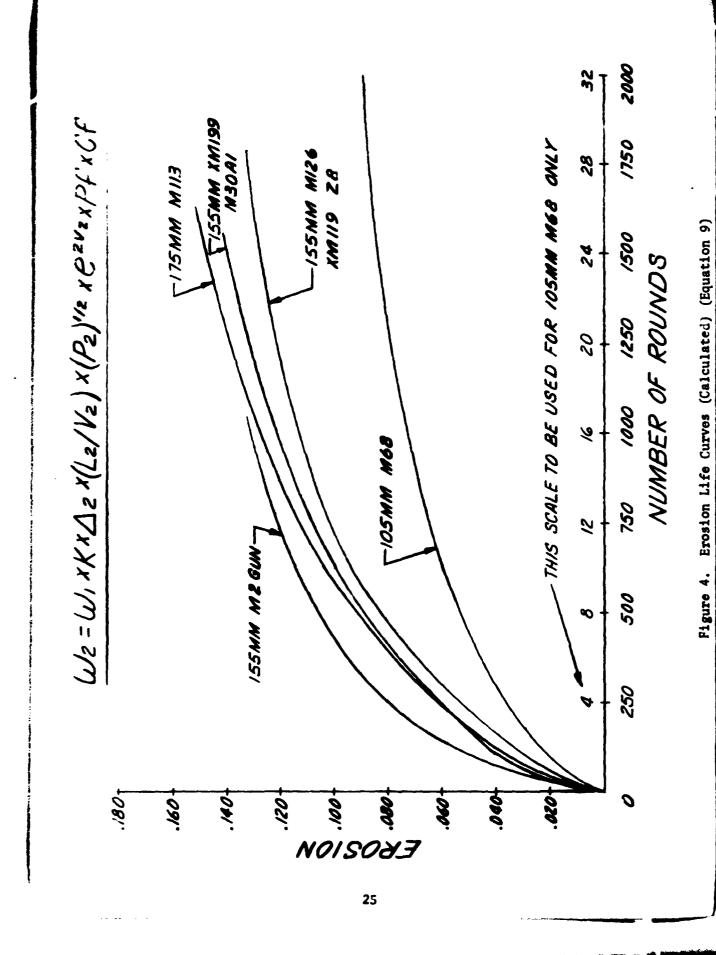
8 Inch XM201 Z8

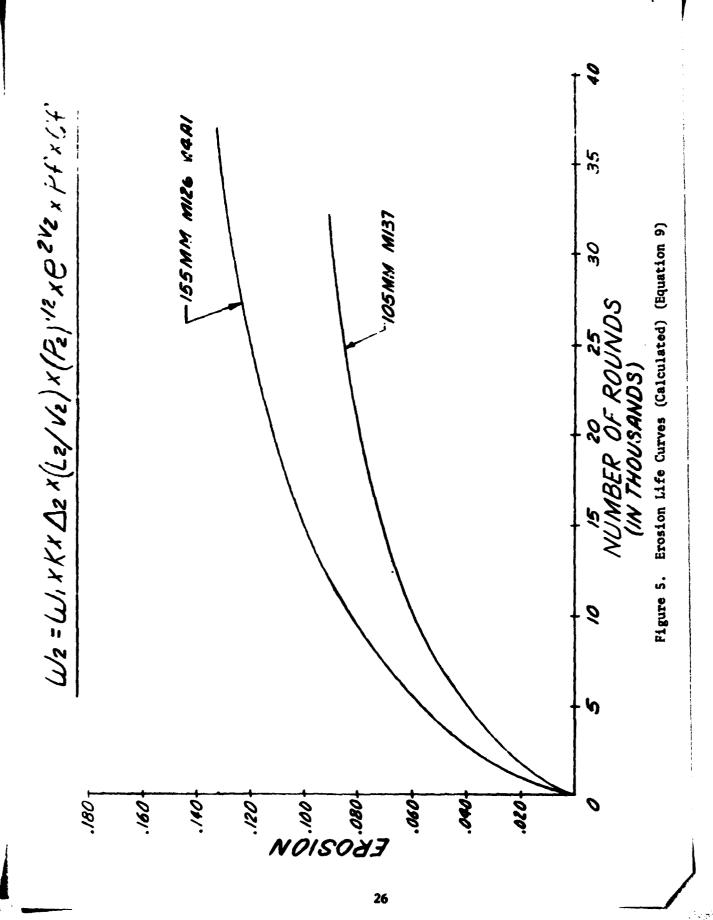
No. of Rds.	Erosion in Inches
362	0.059
751	0.092
1126	0.115
1489	0.131
1850	0.144
2208	0.153
2590	0.159
2957	0.165
3275	0.170
3699	0.177

TABLE 4. COMPARISON WITH FIRING DATA

	Observed Life Max. Wear in Inches at Rounds	Calculate Max. Wear at Rounds	Calculated Life Max. Wear in Inches at Rounds
CANNON		EQUATION 8	EQUATION 9
105mm M137	Fatigue limited	.075/12500	.075/16000
105mm XH205	.04"/2000 (not yet established)	.075/1250	.075/1350
105mm M68 Heat	.075/125	.075/79	.075/16
155mm M126 27	.08"/7500 Patigue limited	.080/8000	080/800
155mm M126 Z8	Not available	0.100/700	0.100/700
155mm XM199 Z8	.150/2300	0.150/3800	0.150/2100 extrapolated
155mm M2 Gun (Normal)	0.110/700	0.110/827	0.110/529
175mm W113	0.200/1200	0.200% 4000 extrapolated	0.200/2400 extrapolated
8" XH201 Z9	0.15/1500 not yet established	0.15/1400	0.15/1200
8" XM201 Z8	0.1/2000 not yet established	0.15/2300	0.15/2200







about double the number of rounds actually observed.

There is a possible explanation for the discrepancy between actual and calculated values of the M113. The time factor for the firing cycle is included in the term L/V. The ratio obtained from this term between the standard gun (M185) and the M113 is about 7/9 i.e., 7ms length of cycle for the M185 and about 9ms for the M113. The data from the spin and setback tables indicates the time of cycle for Z3 M84 charge for M113 is about 21ms. The M185 is not listed in these tables. However, if the time cycle of the M126 XM119 charge is considered to be comparable with the M185, the time of cycle comes to about 9ms. Introducing this correction into the equation yields a value of about 1600 rounds for a wear of 0.2 inches. This is much closer to the observed value of 1200 rounds. On the same basis there will be a slight increase in the number of rounds of XM205 and M68 105mm cannon, but not enough to make the calculated values agree with the observed values.

f. Conclusion

Considering the overall performance, equation (9) seems to give a figure that is within 25% of the observed data, in the case of cannon that have a velocity below 2700 FPS, or better, where the ballistics are not drastically different from the standard tube, i.e., the M185. If the observed or properly calculated time for a firing cycle is introduced, a larger spectrum of weapons can be covered.

Another point to be noted is that the performance of the equation also depends implicitly on the similarity of projectile designs between the standard and any other system. It is possible that a correction for projectile engraving forces may improve the calculated results for the M68 heat round and the XM205 round, where the depth of rifling is only .030 as against 0.050 inches for the standard. But judging from the overall performance it may be stated that equation (9) provides a good estimate of erosion if used judiciously and can indicate, with some confidence, the nature of wear in a completely new cannon system.

In closing, this attempt should only be considered as a beginning of an investigation rather than as a final word. A considerable amount of verification and study is still needed.